

**Amendments to the Specification:**

Please replace paragraphs [0056] and [0057] with the following amended paragraphs:

[0056] The rotating sleeve 15 is movably connected at its end facing away from the spindle nut 10 to a reduction gear 7. This reduction gear 7 as a further part of the drive device 5 is formed as a so-called harmonic drive 13. Such a harmonic drive 13 comprises a flexible toothed sleeve 14, a fixed ring element 69 and a wave generator 18 arranged inside the toothed sleeve 14. The toothed sleeve 14 is releasably connected at its closed side to an end 16 of the rotating sleeve 15. The wave generator 18 is movably connected to a spur gear 8 as a further part of the drive device 5. In particular a rotationally rigid connection between the wave generator 18 and a first spur wheel 19 occurs. This engages at least a second spur wheel 20, whereby both spur wheels 19, 20 form a helical gear 8 and in particular a double helical gear 22. The second spur wheel 20 is rotationally rigidly arranged at a drive shaft 21, whereby two electric motors 9, in particular in the form of a synchronous or asynchronous motor, act on the drive shaft 21.

[0057] For determining the position, in particular of the threaded spindle 11 movable in the longitudinal direction 24, a code carrier 70 of a position sensor 60 is inserted into it at its end assigned to the spindle nut 10 and releasably attached there. Such a code carrier exhibits a position-specific pattern which is scanned by appropriate scanning devices of the position sensor 60 and converted into a corresponding position of the threaded spindle 11.

Please replace paragraph [0063] with the following amended paragraph:

[0063] In the region of the intermediate reservoir 31 or of the cylinder bottom plate 30 a first and a second branch pipe 39, 40 branch from the discharge pipe 34 respectively from the discharge hole 27. The first branch pipe 39 extends up to a pressure switch 41. By means of this pressure switch an electrical signal is given depending on the pressure of the hydraulic fluid in the first branch pipe 39, at least when the hydraulic fluid exceeds a predetermined value. This electrical signal is passed to a servomotor 44, such as for example a stepper motor, for its actuation. Generally, for reasons of redundancy two such servomotors 44, 59 are arranged, refer also to Figure 3. The servomotor 44 exhibits a drive shaft at one end of which a pinion 45 is arranged. This engages a cam disc 46 which is rotationally supported on an outer circumference 56 of the rotating sleeve 15 by means of a roller

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bearing 65. The cam disc 46 exhibits teeth assigned to the pinion 45 and at least an actuating cam 48, which, with appropriate actuation of the servomotor 44 due to the electrical signal of the pressure switch 41, comes into contact with an actuating plunger 47, formed as a roller plunger 49, of the safety valve 42.